Joyce offers Bevel Gear Jacks in several designs including: • Translating

- Keyed for non-rotation
- Keyed for traveling nut (KFTN)
- A guide for ordering is on page 150.

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BEVEL GEAR JACKS ORDERING INFORMATION

Instructions: Select a model number from this chart.

Joyce Bevel Gear [®] Jacks							
BG150S	BG150D						
BG250S	BG250D						
BG375S	BG375D						
BG450S	BG450D						
Follow the design tips (pp. 151-154)							
Detailed product information (pp. 15	5-158).						

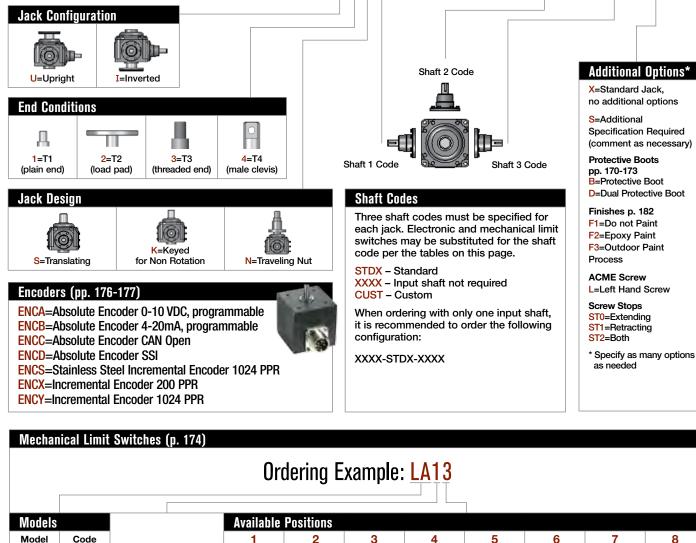
Right hand screw threads standard.

Bevel Gear Jack Rise Rise is travel expressed in inches and not the actual screw length.

Screw Stops (p. 10) and Boots (pp. 170-173)

Screw stops are optional on Bevel gear jacks. When specified, the closed height of the jack and protection tube length may be increased.

Sample Part Number: BG150SU2S-12.25-STDX-STDX-STDX-X





Note: All BG jacks are available with all mounting positions.

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BEVEL GEAR JACKS SPECIFICATIONS AND DESIGN TIPS

Model	Dynamic Capacity	Static Load Capacity		Screw									
		Upright Assembly: screw-in compression/ Inverted Assembly: screw-in tension	Upright Assembly: screw-in tension/ Inverted Assembly: Screw-in compression	Dia.	Pitch/Lead	Bevel Gear Ratio	Pinion Turns for 1" Travel	Pinion Torque (In. Lbs.)	Screw Torque	Jack Efficiency	Jack† Cooling Time	Base Weight (Lbs.)	Add for Each Inch of Travel (Lbs.)
BG150-S	Please use JAX® Online software or contact Joyce	14.000 lbs	14.000 lbs	1 1/2"	.375P STUB ACME	2.69:1	7.18	.059W*	.151W*	38.5%	38 min.	42	.8
BG150-D*		14,000 lbs.	14,000 lbs.		.250P / .500L ACME 2C	2.69:1	5.38	.066W*	.169W*	45.6%	38 min.	42	.8
BG250-S		30,000 lbs.	30,000 lbs.	2 1/2"	.500P ACME 2C	2.15:1	4.31	.111W*	.227W*	34.2%	82 min.	140	2.6
BG250-D*					.375P / .750L ACME 2C	2.15:1	2.87	.133W*	.272W*	42.6%	82 min.	140	2.6
BG375-S		66.000 lba	40,000 lbs 0,0/4	3 3/4"	.666P ACME 2C	3.52:1	5.29	.098W*	.329W*	31.5%	192 min.	230	4.1
BG375-D*		66,000 lbs.	40,000 lbs.	3 3/4	.666P / 1.333L STUB ACME	3.52:1	2.64	.134W*	.448W*	46.0%	192 min.	230	4.1
BG450-S		218,000 lbs. 200,000 lbs. 4 1/	000 000 lba	4 1 /0"	.500P ACME 2C	3:1	6	.125W*	.356W*	21.9%	262 min.	650	5.5
BG450-D*			.500	.500P / 1.00L ACME 2C	3:1	3	.154W*	.438W*	35.5%	262 min.	650	5.5	

Important Note: *Not self-locking, may lower under load. Brake motors or external locking systems are recommended.

D: Double Lead Screws.

S: Single Lead Screws. These jacks are self-locking.

*W: Load in Pounds.

Pinion Torque: The torque required to continuously raise a given load.

Screw Torque: The torque required to resist screw rotation (translating jack design) and traveling nut rotation (keyed for traveling nut design).

Lead: The distance traveled axially in one rotation of the lifting screw.

Pitch: The distance from a point on the screw thread to a corresponding point on the next thread, measured axially.

†: Cooling time based on time to cool from 200°F to 70°F (ambient).

Design Tips:

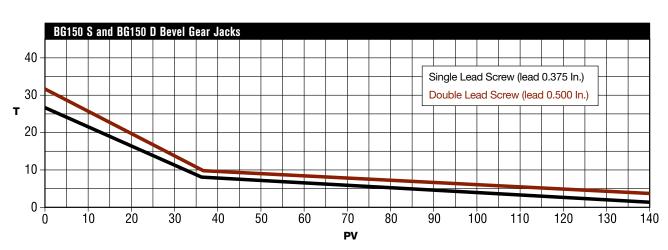
- A PV (pressure/velocity) value must be calculated for each application. The continuous running time should not exceed the corresponding T (time) value. Refer to instructions and graphs on pages 152 and 153.
- Cooling time data on these charts is calculated based on limiting the lifting nut temperature rise from 70°F to 200°F (100° below dropping point of grease).
- Check single lead versus double lead screws in each case. A double lead screw may be the appropriate choice due to increased efficiency while offering the same performance characteristics.
- 4. JAX[®] Online software is a useful design aid to determine the following:
 - The allowable static compression load for a given rise (or use Column Loading Chart on page 154)
 - The allowable dynamic load for a given rise
 - System horsepower and torque also see item #5
- 5. When a direct motor drive is used in a system, consideration must be given to the input starting torque requirements and the motor horsepower will need to be increased accordingly (JAX[®] Online software data may require additional scrutiny). Contact Joyce for assistance.
- 6. When selecting multiple bevel gear jacks for an interconnected row or system (page 195) careful attention must be given to the input and output shaft rotations. For example, if the input shaft rotation on the first jack is clockwise, the output shaft(s) on that same jack will rotate counter-clockwise. To insure all jacks raise and lower in unison, alternating jacks must be specified with right and left hand acme screw threads. For example, if you have five jacks interconnected in a straight line and the first jack is right hand, the third and fifth jack will also need to be ordered as left hand. Bevel gear jacks are supplied standard with right and acme screws. To order the left hand acme screw option, add an "L" to the end of your bevel gear jack part number as shown on page 150.
- Joyce Bevel Gear[®] "S" Series (single lead) jacks are inherently self-locking. A brake is required for "D" series (double lead) jacks, which may lower under load.
- Bevel gear jacks are furnished with one input shaft in position #2. Jacks may be ordered with up to three input shafts located at any combination of positions # 1, 2, or 3.
- Joyce Bevel Gear[®] jacks are designed for oil bath (EP-90 gear lubricant) or grease operation. The upper bearing is grease lubricated through a fitting on top of the jack. Grease must be applied directly to the lifting screw.
- 10. Typically jacks are mounted upright with the base plate parallel to the horizon. If the base plate is oriented any other way, contact Joyce for lubrication and other instructions.

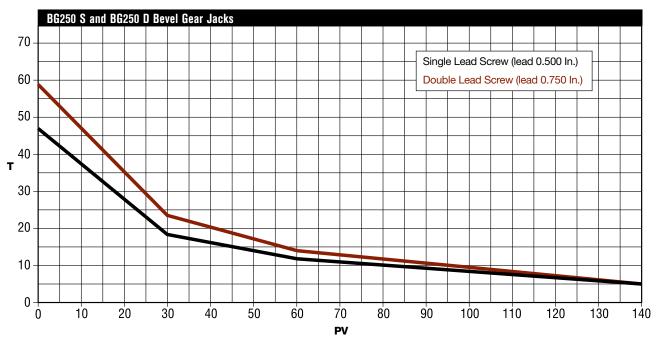
BEVEL GEAR JACKS APPLICATION INFORMATION AND THERMAL GRAPHS

In many applications, Joyce Bevel Gear[®] jacks are more efficient and faster than wormgear driven jacks. To determine the suitability of a bevel gear jack for your application, use the steps below to calculate load, travel speed and duty cycle.

- **Step 1** Determine load in pounds.
- Step 2 Determine velocity in feet / minute (fpm).
- Step 3 Determine duty cycle in terms of minutes operation / minutes resting (or time on / time off).
- Step 4Calculate PV.PV = (load x velocity in fpm)/1000
- **Step 5** Calculate cooling time (T). T = Cooling time (p. 151) x $\frac{\text{time on}}{\text{time off}}$

- **Step 6** Plot the points for PV and T on the appropriate graph (below or on the next page). If the point falls below the line, the application is satisfactory. If it is above the line, recalculate T for the next larger size jack. Each jack size has a different cooling time (p. 151).
- **Step 7** Calculate horsepower. RPM = Velocity in fpm x 12 x input turns per one-inch travel (from chart on p. 151) Horsepower = Pinion torque (from chart) x load x RPM





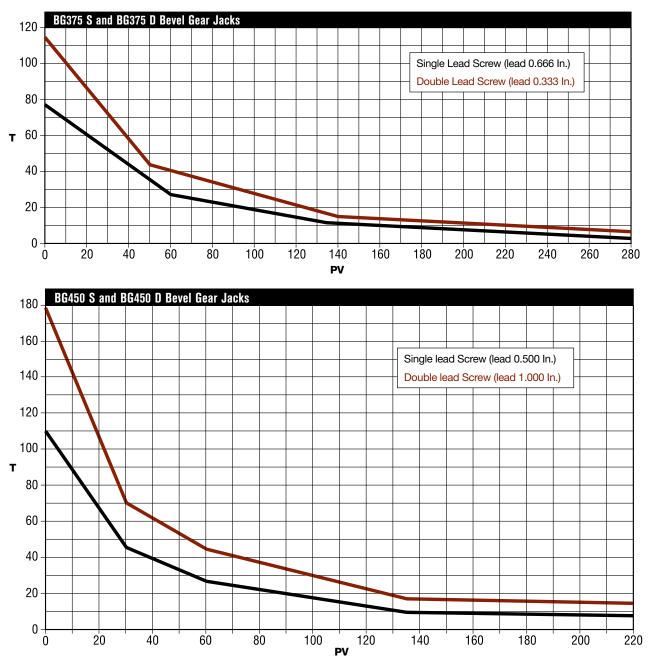
Note: PV = load x velocity (fpm)1000 \mathbf{T} = the maximum running time in minutes before a complete cooling time is required.

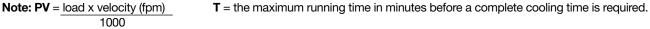
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BEVEL GEAR JACKS EXAMPLE AND THERMAL GRAPHS

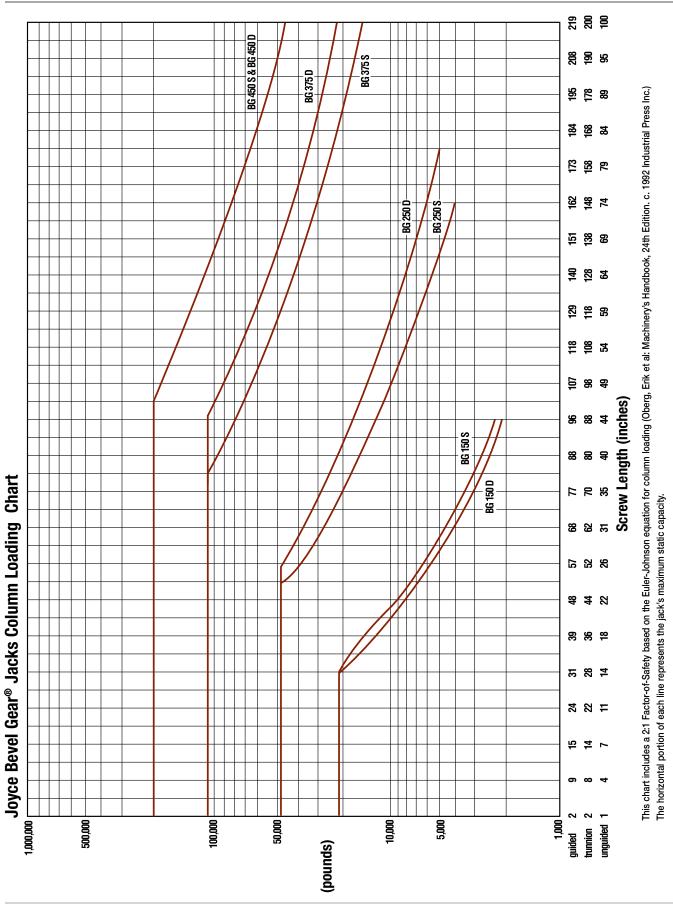
Example: A 5000-pound load must be raised 30 inches in 15 seconds. The load remains in position for two minutes. It is then lowered and remains lowered for 30 seconds. The cycle begins again. Determine the appropriate bevel gear jacks and calculate horsepower required.

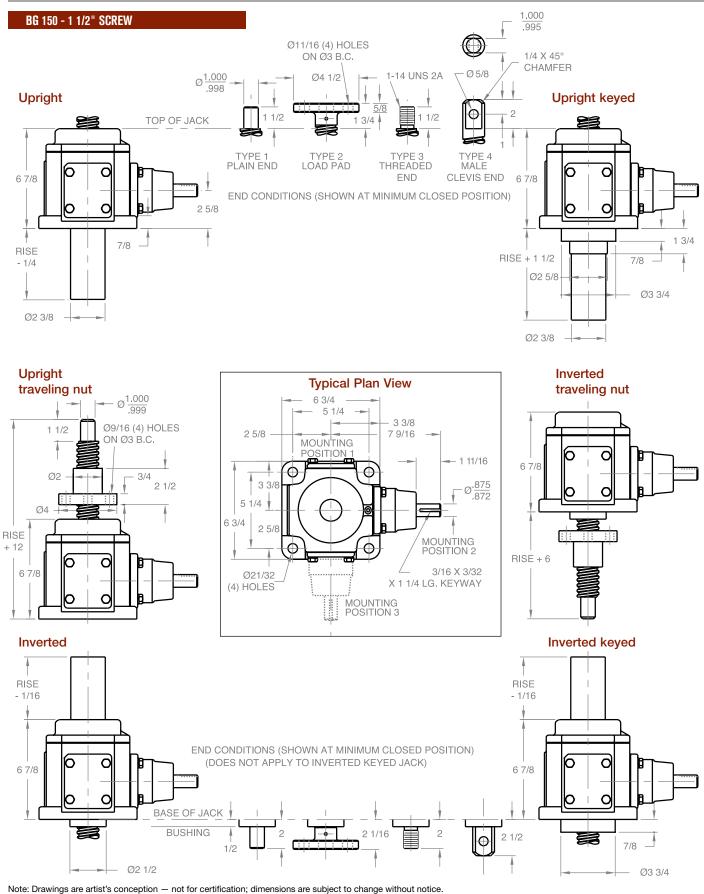
- **Step 1** Load = 5000 pounds
- **Step 2** Velocity = 30 inches in 15 seconds = 10 fpm
- Step 3 Duty cycle = Time on / Time off
 Time on = 15 seconds up +15 seconds down = 30 seconds = 0.5 min
 Time off = 2 minutes up + 30 seconds down = 2 minutes 30 seconds = 2.5 minutes
- Step 4 PV= (5000 x 10) / 1000 = 50
- **Step 5** T = 38 (for BG150) x (0.5 / 2.5) =7.6
- **Step 6** The point for PV, 50. and T, 7.6 falls below the line for BG 150 D and above the line for BG 150S, therefore BG 150 D is appropriate. (reference BG150 chart on p. 152)
- **Step 7** RPM = 10 x 12 x 5.38 = 645.6 Horsepower = (.066 x 5000 x 646) / 63,025 = 3.38



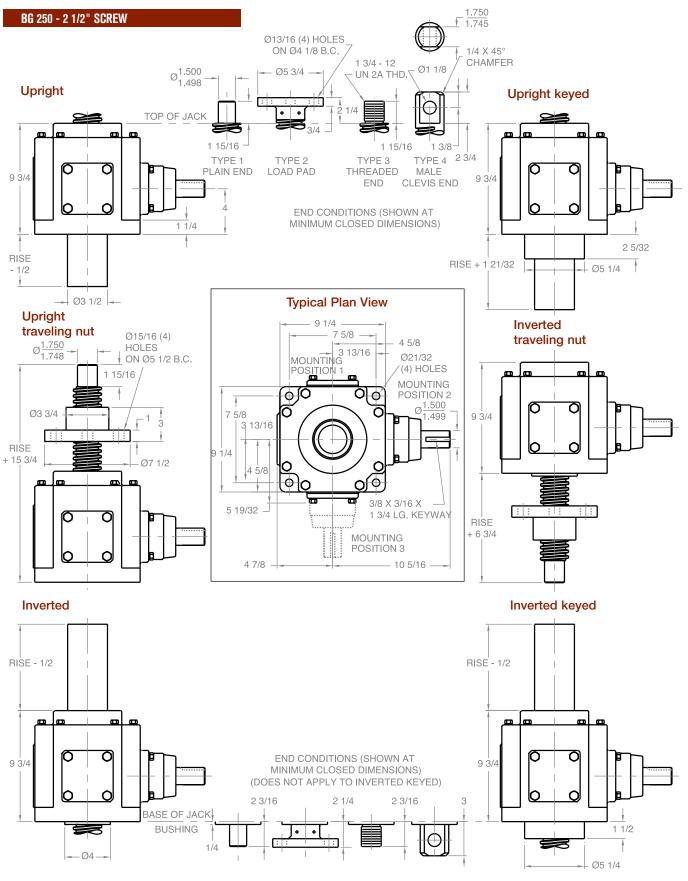


BEVEL GEAR JACKS COLUMN LOADING

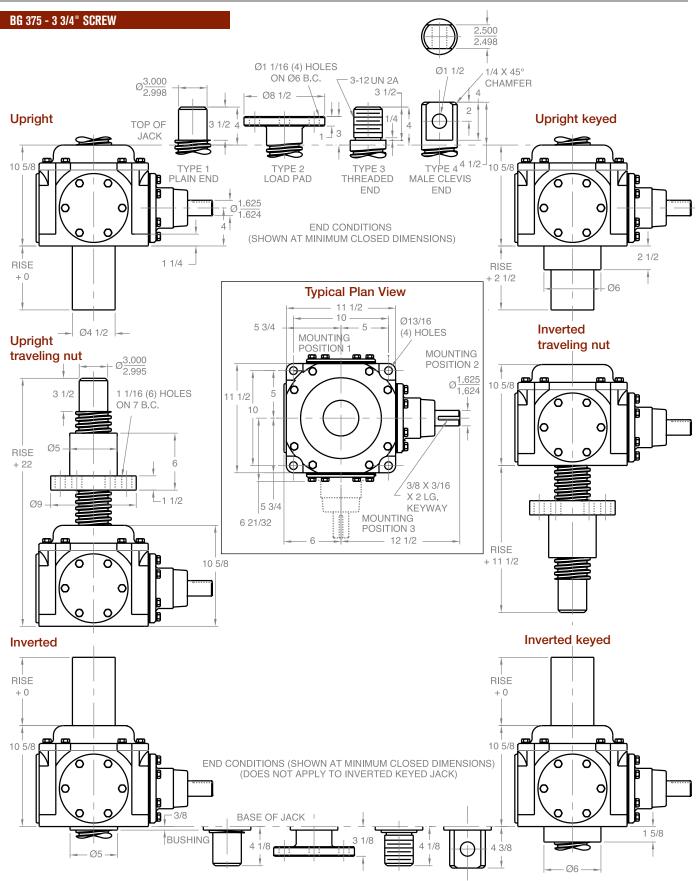




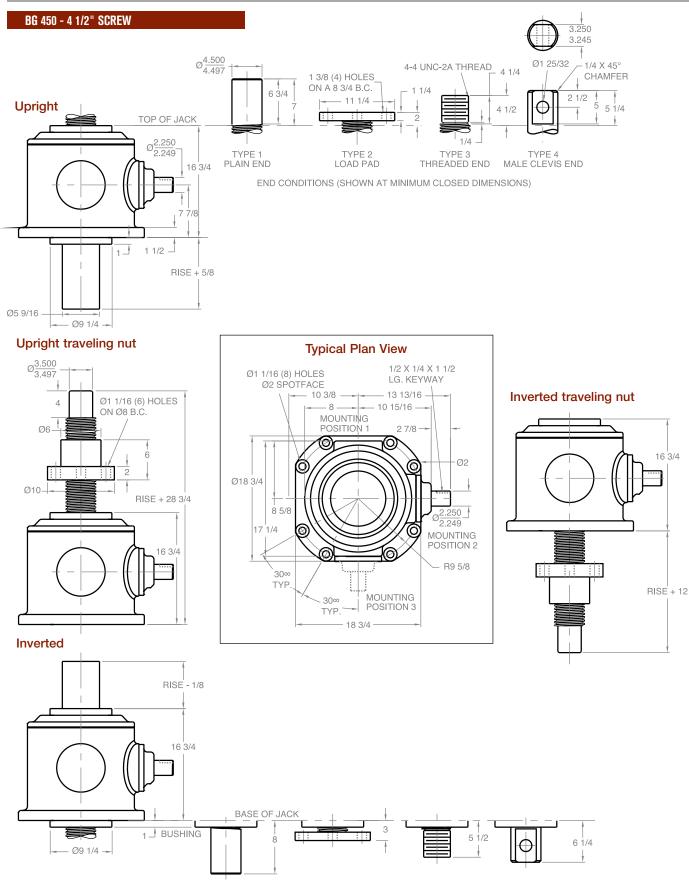
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Note: Drawings are artist's conception - not for certification; dimensions are subject to change without notice.



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